

**REMARKS**

New Claims 13 through 24 have been added and are currently pending in the present application. Original claims 1 through 12 have been cancelled. In view of the above amendment, applicant believes the pending application is in condition for allowance.

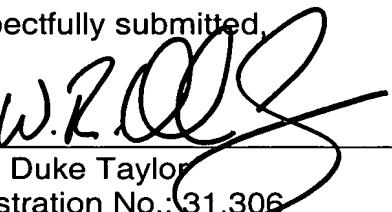
Thus, prompt and favorable consideration of this amendment is respectfully requested. If the Examiner believes that personal communication will expedite prosecution of this application, the Examiner is invited to telephone the undersigned at (248) 641-1600.

Applicant believes no fee is due with this response. However, if a fee is due, please charge our Deposit Account No. 08-0750, under Order No. 6340-000075/NP from which the undersigned is authorized to draw.

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Respectfully submitted,

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## A BEARING APPARATUS FOR A WHEEL OF VEHICLE

### CROSS-REFERENCE TO RELATED APPLICATIONS

[0001] This application is a National Stage of International Application No. PCT/JP2004/015144, filed October 14, 2004, which claims priority to Japanese Patent Application No. 2003-401112, filed December 1, 2003, Japanese Patent Application No. 2004-0079683, filed March 19, 2004 and Japanese Patent Application No. 2004-269093, filed September 16, 2004. The disclosures of the above applications are incorporated herein by reference.

### FIELD

#### Field of the Invention

[0002] —The present disclosure invention relates to a vehicle wheel bearing apparatus for a wheel of vehicle for rotatably supporting the wheel relative to a suspension apparatus of the vehicle, and, more particularly, to a vehicle wheel bearing apparatus for a wheel of vehicle of the semi-floating type wherein which a driving wheel is supported by a double row rolling bearing.

### BACKGROUND

#### Description of Background Art

[0003] —In a vehicle such as a truck having a body with a frame structure, an axle structure of a driving wheel of a full-floating type bearing has been widely adopted. In a recent driving wheel supporting structure, a unit structure of a double row rolling bearing there has been widely adopted a unit structure of a double row rolling bearing. This unit as to improve improves the readiness of assembly,

and reduction of weight and size. One example of such a prior art vehicle wheel bearing apparatus ~~for a wheel of vehicle of the prior art~~ is shown in Fig. 9.

**[0004]** In this vehicle wheel bearing apparatus ~~for a wheel of vehicle~~, a drivingdrive shaft 52, connected to a differential apparatus (not shown), is inserted into an axle housing 51. A and a double row conical roller bearing 53 is mounted on the axle housing 51. A hub-wheel hub 54 is rotatably supported by the double row conical roller bearing 53. The wheel hub 54 is connected to a flange 56 via hub bolts 55. A pair of inner rings 57 is are connected to each other by a connecting ring 58. The rings 57 are and fitted onto the end of the axle housing 51 and then securely fastened by a fastening nut 59. On the other hand anAn outer ring 60 of the double row conical roller bearing is fitted into the hub-wheelwheel hub 54. The outer ring 60 is and axially secured bywith its both its ends being sandwiched by the flange 56 of the drivingdrive shaft 52 and a brake rotor 61. A doubleDouble row conical rollers 62 are rollably contained by cages 63 between the annular space between the inner and outer rings 57 and 60. Seals and seals 64 are arranged at both ends of the annular space to seal off the inside of the wheel bearing off from the outside.

**[0005]** The inboard side end of the inner ring 57 is formed with an annular stepped portion 65. The stepped portion 65 receives and mounts ~~on which~~a seal ring 66 ~~is mounted~~. An annular recess 67 is formed on the outer circumferential surfaces of the inner rings 57 at mutually abutted portions of the pair of inner rings 57. An elastic and a seal ring 68 ~~of elastic member~~ is fitted into the recess 67~~therein~~. These seal rings 66 and 68 prevent penetration or ingress of rain water or dusts into the axle housing 51, leakage of differential gear oil to the outside and

ingress of the differential gear oil into the inside of the bearing (see Japanese Laid-open Patent publication No. 99172/2001).

Disclosure of the Invention

Problems to be solved by the Invention

[0006] However since the prior art vehicle wheel bearing apparatus for a wheel of vehicle of the prior art has a structure wheresuch that the double row of conical roller bearing 53 is arranged between the hub-wheelwheel hub 54 and the axle housing 51, and that the driving shaft 52 is inserted into the axle housing 51 and then the flange 56 of this drivingdrive shaft 52 is connected to the hub-wheelwheel hub 54 by the hub bolts 55, a reduction of the weight and size of the bearing apparatus is limited. Also, the as well as assembly of the bearing apparatus is complicated bydue to the requirement of a large number of structural parts.

SUMMARY OF THE INVENTION

[0007] It is therefore an object of the present disclosureinvention to provide a vehicle wheel bearing apparatus for a wheel of vehicle which can reduce the weight, size and atthe number of parts. Also, a bearing apparatus and also can prevent the ingress of rain water or dusts and the leakage of differential gear oil.

[0008] For achieving the object, there is provided, accordingAccording to the present disclosure, a vehicle wheel invention of claim 1, a bearing apparatus for a wheel of vehicle comprising:comprises an axle housing supported under a body of a vehicle. A: a hollow drivingdrive shaft is inserted into the axle housing. A: and a wheel bearing is arranged between the drivingdrive shaft and an opening of the axle housing and is structured as a unit of a hub-wheelwheel hub and a double row rolling

bearing. The; the wheel bearing comprising:comprises an inner member which includesincluding a hub-wheelwheel hub with an integrally formed wheel mounting flange on one end thereof with a wheel mounting flange and having an axially extending cylindrical portion. At least one inner; and inner rings is press-fitted fit onto the cylindrical portion of the hub-wheelwheel hub. The and formed on which outer circumferential surface of the inner ring is formed with at least one of the inner raceway surfaces. An; an outer member is arranged around the inner member. The outer member is and formed with double row outer raceway surfaces on its inner circumferential surface. The outer raceway surfaces are oppositelyopposite to the inner raceway surfaces. Double; double row rolling elements are arranged between the inner and outer raceway surfaces of the inner member and the outer member. A; a cage for freely rollably holding holds the rolling elements. Seals are provided to ; and seals for sealing seal an annular space between the inner member and the outer member. A cap, with a ; characterized in that a cap having metal core formed from steel, is press-fitted fit into an end of a central bore of the hub-wheelwheel hub.

[0009] According to the present invention of claim 1, sinceSince at the cap, with the steel having metal core of steel, is press-fitted into an end of the central bore of the hub-wheelwheel hub forming the wheel bearing apparatus, it is possible to provide a vehicle wheel bearing apparatus for a wheel of vehicle of a semi-floating type which can reduce the weight and size. Also, the bearing apparatus and also can prevent the leakage of differential gear oil to the outside as well as prevent the ingress of rain water or dusts from the outside into the differential gear oil through the drivingdrive shaft.

[0010] According to the present invention of claim 2, since Since said at least one of the inner raceway surfaces is formed directly on the outer circumferential surface of the hub-wheelwheel hub, it is possible to further reduce the bearing weight and size and increase the rigidity of the bearing.

[0011] According to the present invention of claim 3, since Since the end of thesaid cylindrical portion is radially outwardly plastically deformed radially outward to form a caulked portion ~~for to preventing~~ prevent the inner ring from being slippedslipping off from the cylindrical portion of the hub-wheelwheel hub, it is unnecessary to control the amount of preload of the bearing as in the prior art, by tightly fastening a nut on the inner ring using a nut. Thus, the ease easiness of assembly of the bearing apparatus ~~to onto~~ a vehicle can be improved. Also, and the predetermined amount of preload can be kept for a long term. In addition, it is possible to substantially reduce the number of parts and to reduce ~~the~~ manufacturing cost, and the weight and size of the bearing due to the improvements in~~of~~ the easinessease of assembly.

[0012] Preferably according to the present invention of claim 4, since Since the outer circumferential region of the wheel mounting flange, from its base to the inboard side to the cylindrical portion, is hardened by high frequency induction hardening to have~~as~~ having the surface hardness of 54~64 HRC, and the caulked portion is remained~~remains~~ unhardened to have~~as~~ having the surface hardness of 25 HRC or less after forging, it is possible to improve the durability of the hub-wheelwheel hub and workability of the caulked portion during its plastic deformation. Thus, and thus the reliability of the quality of the bearing is improved.

[0013] According to the present invention of claim 5, since the said cap is press-fitted fit into the central bore of the wheel hub wheel mounting flange of hub wheel, the cap can be positioned at a region with having high rigidity of the hub wheelwheel hub. Accordingly, the cap is scarcely influenced by elastic deformation of the hub wheelwheel hub and thus, it is possible to prevent the generation of a radial gap between the cap and the hub wheelwheel hub.

[0014] According to the present invention of claim 6, since Since said the cap comprises includes a metal core made of steel with having a cross-section of substantially "C" shaped configuration cross-section and an elastic member is attached to at least part of its fitting portion, the elastic member can intimately contact the fitting surface and thus can securely seal the inside of the hub wheelwheel hub.

[0015] According to the present invention of claim 7, since Since said the cap is press-fitted fit so that the circumferential edge of its fitting portion is oriented toward the outboard side, the press-fitting operation can be easily carried out. In addition, since the edge side of low rigidity is positioned at the outboard side, the cap does not move toward the outboard side. Thus, and thus slipping off of the cap from the hub-wheelwheel hub can be prevented even although though the metal core is moved axially due to its deformation caused by elastic deformation of the hub wheelwheel hub.

[0016] According to the present invention of claim 8, since Since the said circumferential edge of the fitting portion of the metal core is formed with ana bead

extending radially outward bead, and an annular groove ~~with which engages~~ the bead ~~engages~~ is formed on the central bore of the ~~hub-wheel~~wheel hub, it is possible to securely prevent the axial movement of the cap. ~~This and thus to further improve~~improves the reliability of the cap.

[0017] ~~According to the present invention of claim 9, since the said cap is limited against an axial movement by steps provided at either sides side of the cap, it is possible to securely prevent the cap being from slipping off from the hub-wheel~~wheel hub even though~~although~~ the ~~hub-wheel~~wheel hub is elastically deformed by a repeating load applied to it thereto during running of a vehicle.

[0018] ~~According to the present invention of claim 10, since Since the said cap comprises~~includes a metal core made of steel ~~with~~having a cross-section of substantially "C" shaped configuration cross section, an annular recess is formed on the inner circumferential surface of the ~~hub-wheel~~wheel hub, and the fitting portion of the cap is formed with a projection adapted to be engaged with the annular recess, it is possible to easily mount the cap on the ~~hub-wheel~~wheel hub and to prevent the axial movement of the cap with a simple structure.

[0019] ~~Preferably according to the present invention of claim 11, since Since the said~~ projection is formed by plastic deformation after the cap has been press-fitted fit into the bore of the ~~hub-wheel~~wheel hub, the cap can be further intimately fitted into the annular groove of the ~~hub-wheel~~wheel hub without any rattle. ~~Thus, and thus the axial movement of the cap and also therefore the leakage of differential gear oil can be further prevented by the mating of the projection and bore.~~

[0020] Since ~~Preferably~~ according to the present invention of claim 12, since the said cap is press-fitted with an interference of 0.05~0.3 mm, it is possible to prevent the generation of the radial gap between the cap and the hub wheelwheel hub due to errors in configuration of the cap. Thus, this prevents and thus to prevent the leakage of differential gear oil therethrough. In addition, the cap can be easily press-fitted fit into the hub wheelwheel hub and the buckling of the cap, which would be caused by large interference, can also be also prevented.

Effect of the Invention

[0021] According to the The vehicle wheel bearing apparatus for a wheel of vehicle of the present inventionand disclosure, which, since it comprises an axle housing supported under a body of vehicle; a hollow drivingdrive shaft inserted into the axle housing; and a wheel bearing arranged between the drivingdrive shaft and an opening of the axle housing which is and structured as a unit of a hub wheelwheel hub and a double row rolling bearing; the wheel bearing comprising:comprises an inner member withincluding a hub wheelwheel hub integrally formed with a wheel mounting flange on one end thereof with a wheel mounting flange and having an axially extending cylindrical portion; at least one and inner rings is press-fitted fit onto the cylindrical portion of the hub wheelwheel hub and is formed on itswhich outer circumferential surface with at least one of the inner raceway surfaces; an outer member is arranged around the inner member and is formed with double row outer raceway surfaces on its inner circumferential surface oppositelyopposite to the inner raceway surfaces; double row rolling elements are arranged between the inner and outer raceway surfaces of the inner member and the outer member; a cage forfreely rollably holdingholds the rolling elements; and seals forsealingseal an annular space

between the inner member and the outer member; and it is characterized in that a cap, with a having metal core of steel, is press-fitted fit into an end of a central bore of the hub wheelwheel hub, makes it is possible to provide a vehicle wheel bearing apparatus of for a wheel of vehicle of a semi-floating type which can reduce the weight and size of the bearing apparatus. Also, it and also can prevent prevents the leakage of differential gear oil to the outside as well as the ingress of rain water or dusts from the outside into the differential gear oil through the drivingdrive shaft.

Best mode for carrying out the Invention

[0022] According to the disclosure represent invention, there is provided a vehicle wheel bearing apparatus for a wheel of vehicle comprisingcomprises: an axle housing supported under a body of the vehicle. A; a hollow drivingdrive shaft is inserted into the axle housing. A; and a wheel bearing is arranged between the drivingdrive shaft and an opening of the axle housing. The wheel bearing is and structured as a unit with a hub wheelwheel hub and a double row rolling bearing. The; the wheel bearing hascomprising: an inner member which includesincluding a hub wheelwheel hub integrally formed with a wheel mounting flange on one end thereof with a wheel mounting flange and having an axially extending cylindrical portion. At least one inner ring is; and inner rings press-fitted fit onto the cylindrical portion of the hub wheelwheel hub. The inner ring and formed on which outer circumferential surfaceouter circumferential surface is formed with at least one of inner raceway surfaces. An; an outer member is arranged around the inner member. The outer member is and formed with double row outer raceway surfaces on its inner circumferential surface oppositelyopposite to the inner raceway surfaces. Double; double row rolling elements are arranged between the inner and outer raceway

surfaces of the inner member and the outer member, A; a cage for freely rollably ~~holding~~holds the rolling elements, Seals; and ~~seals~~seal for sealing an annular space between the inner member and the outer member, A; characterized in that a cap, with a having metal core of steel, is press-fitted into an end of a central bore of the ~~hub-wheel~~wheel hub.

#### BRIEF DESCRIPTION OF THE DRAWINGS

**[0023]** Additional advantages and features of the present ~~disclosure~~invention will become apparent from the subsequent description and the appended claims, taken in conjunction with the accompanying drawings, wherein:

**[0024]** Fig. 1 is a longitudinal-section view of a first embodiment of a ~~vehicle wheel~~ bearing apparatus ~~for a wheel of vehicle of the present invention~~;

**[0025]** Fig. 2 is a partially enlarged longitudinal-section view of Fig. 1 showing a wheel bearing;

**[0026]** Fig. 3 is a longitudinal-section view of a second embodiment of a ~~vehicle wheel~~ bearing apparatus ~~for a wheel of vehicle of the present invention~~;

**[0027]** Fig. 4 is a partially enlarged longitudinal-section view of Fig. 3;

**[0028]** Fig. 5 is a longitudinal-section view of a third embodiment of a ~~vehicle~~ bearing apparatus ~~for a wheel of vehicle of the present invention~~;

**[0029]** Fig. 6 is a longitudinal-section view of a fourth embodiment of a ~~vehicle wheel~~ bearing apparatus ~~for a wheel of vehicle of the present invention~~;

**[0030]** Fig. 7(a) and 7(b) are a partially enlarged longitudinal-section view of a modification of the fourth embodiment showing, respectively, a condition of a cap before and after caulking ~~thereof~~;

[0031] Fig. 8 is a longitudinal-section view of a fifth embodiment of a vehicle wheel bearing apparatus ~~for a wheel of vehicle of the present invention; and~~

[0032] Fig. 9 is a longitudinal-section view of a prior art vehicle wheel bearing apparatus ~~for a wheel of vehicle of the prior art.~~

#### DESCRIPTION OF THE PREFERRED EMBODIMENTS

[0033] Preferred embodiments of the present disclosure ~~invention~~ will be described with reference to the accompanied ~~accompanying~~ drawings.

##### First embodiment

[0034] Fig. 1 is a longitudinal-section view of a first embodiment of a vehicle wheel bearing apparatus ~~for a wheel of the present invention, and~~ Fig. 2 is a partially enlarged longitudinal-section view of Fig. 1. In the present ~~description of the present invention~~, a side of the bearing positioned outward of ~~for~~ a vehicle, when it is mounted on the vehicle, is referred to as the "outboard" side (the left side in a drawing), A, and a side inward of the vehicle is referred to as the "inboard" side (the right side in a drawing).

[0035] In the vehicle wheel bearing apparatus ~~for a wheel of vehicle of the present invention~~, a hub wheel ~~wheel hub~~ 1 and a double row rolling bearing 2 are formed as a unit and are connected to a driving ~~drive~~ shaft "D/S". The double row rolling bearing 2 includes ~~comprises~~ an inner member 3, an outer member 4, and a double row rolling elements (tapered rollers) 5 freely rollably contained between the inner and outer members 3 and 4. The inner member 3 includes the hub wheel ~~wheel hub~~ 1 and a pair of inner rings 10 press-fitted fit onto the hub wheel ~~wheel hub~~ 1. The

~~hub wheel~~ hub 1 is integrally formed, at its outboard side, with a wheel mounting flange 6 on which, a wheel "W" and a brake rotor "B" are mounted. An ~~and from which an~~ axially extending cylindrical portion 7 extends from the flange 6. An inner circumferential surface (bore) of the ~~hub wheel~~ hub 1 is formed with a serration (or spline) 8 ~~to receive into which~~ a serrated portion of the ~~driving~~ drive shaft "D/S" ~~to transmit~~ is inserted so that a torque between the two can be transmitted therebetween.

[0036] As shown in Fig. 2, the double row rolling bearing 2 ~~includes~~ comprises an outer member 4 formed with double row outer raceway surfaces 4a on its inner circumferential surface. A ~~and with a~~ body mounting flange 4b is formed on the outer member 4. The flange 4b is to be secured on an axle housing "H" on its outer circumferential surface. A, a pair of inner rings 10 are inserted into the outer member 4. The inner rings 10 are formed with double row tapered inner raceway surfaces 10a, 10a on ~~its~~ their outer circumferential surface ~~opposite~~ oppositely to the outer raceway surfaces 4a. Double, double row rolling elements 5 are arranged between the inner and outer raceway surfaces 10a, 4a. A, and a cage 11 ~~for~~ freely rollably ~~holding~~ holds the rolling elements 5. Each of the inner rings 10 is formed with, at its larger diameter end, a large flange 10b to ~~guide~~ for guiding the rolling elements 5. The pair of inner rings 10 are arranged so that their inner ends are abutted each other. Thus, they and thus form a ~~seso~~-called a back-abutted type double row tapered roller bearing. Seals 12 are arranged at ~~bo~~ either ends of the outer member 4 to seal an annular space between the outer member 4 and the inner rings 10. These ~~The~~ seals 12 prevent both penetration of

rain water or dusts from the external circumstances and leakage of lubricating grease sealed within the bearing. The inboard side seal 12 further prevents penetration or ingress of differential gear oil into the inside of the bearing.

[0037] The pair of inner rings 10 are press-fitted onto the cylindrical portion 7 of the hub-wheelhub 1. The inner rings 10 are prevented from being axially slipped off of from the cylindrical portion 7 by a caulked portion 13. The caulked portion is formed by plastically deforming the end of the cylindrical portion 7 radially outward. Since this embodiment adopts the self-retaining structure of the second generation, it is not required to control an amount of preload as in a conventional manner by tightly fastening a nut against the inner ring. Accordingly, it is possible to substantially reduce the number of parts and thus to improve the readiness of assembly as well as to reduce its manufacturing cost, size and weight and size.

[0038] The hub-wheelhub 1 is made of medium carbon steel such as S53C which includes carbon of 0.40~0.80% by weight. The wheel hub 1 is hardened by high frequency induction quenching so that the base of the wheel mounting flange 6, at its inboard side, and the cylindrical portion 7 of the hub-wheelhub 1 have at the surface hardness of 54~64 HRC (the hardened portion is shown in drawings by cross-hatched lines). The caulked portion 13 is remained as an unhardened portion and has having its surface hardness of 25 HRC or less. This improves the durability and workability of the caulked portion 13 and also prevents the generation of cracks therein.

[0039] The outer member 4 is also made of medium carbon steel such as S53C which includes including carbon of 0.40~0.80% by weight. The and the double row outer raceway surfaces 4a and inner circumferential surface of the outer member 4, on which the seal 12 is mounted, are hardened by high frequency induction quenching so that their surface hardness is within 54~64 HRC. On the other hand, the The inner rings 10 are made of high carbon chrome bearing steel such as SUJ2. The inner rings 10 are and hardened to the its core by dipping dip quenching to have thea surface hardness of HRC 54~64. Even Although though the it is herein illustrated a double row tapered roller bearing is illustrated using using tapered roller as the rolling elements 5, thea double row angular ball bearing using balls may be also be used.

[0040] In this embodiment, a cap 9 is press-fitted into an opening of the hub wheelwheel hub 1 at its outboard side. This cap 9 is made of austenitic-stainless steel sheet (JIS SUS 304 etc.) or preserved cold rolled steel sheet (JIS SPCC etc.) and is formed as an annular shape by press working. The cap 9 comprises includes a metal core 9a formed from of steel. The cap 9 has formed as having a substantially "C"-shaped cross-section. An, and an elastic member 9b, of rubber, is bonded, via vulcanization, to at least the fitting portion of the metal core 9a. The elastic member 9b is elastically deformed during the press fitting of the cap 9 is press-fitted into the opening of the hub wheelwheel hub 1 to seal the opening. The seal for surely preventing prevents ingress of rain water or dusts from the ambient circumstances into the drivingdrive shaft "D/S" and thus into the differential gear oil.

[0041] It is preferable that the cap 9 is press-fitted into the hub wheelhub 1 with a interference of 0.05~0.3 mm. This is because ~~the~~ differential gear oil could leak through a small radial gap which would be caused in the fitting portion between the hub wheelhub 1 and the cap 9 due to dimensional errors of the cap itself when the interference is less than 0.05 mm. On, ~~or~~ the other hand, the press-fitting operation of the cap 9 would become~~becomes~~ difficult and buckling of the metal core may occur itself would be caused when the interference is larger than 0.3 mm. In addition, it is preferable that the cap 9 is press-fitted into the hub wheelhub 1 at a high rigid bore portion. That is, ~~thereof~~, ~~that is~~ a bore portion of the hub wheelhub 1 at or near the wheel mounting flange 6. Accordingly, the cap 9 is scarcely influenced by elastic deformation of the hub wheelhub 1 although~~even though~~ the hub wheelhub 1 would be deformed by applying repeating repeated moment loads applied thereto.

#### Second embodiment

[0042] Fig. 3 is a longitudinal-section view of a second embodiment of the vehicle wheel bearing apparatus ~~for a wheel of vehicle of the present invention~~. Since the difference between of this embodiment and from the first embodiment only resides in the structure of the hub wheelhub, the same numerals are used as those used in the first embodiment for designating to designate the same structural elements.

[0043] ~~This~~The vehicle wheel bearing apparatus ~~for a wheel of vehicle~~ is structured as a unit with the of the hub wheelhub 14 and a double row rolling bearing 15. The double row rolling bearing 15 comprises~~includes~~ an inner member

16, an outer member 4, and a double row rolling elements 5 and 5 freely rollably contained between the inner and outer members 16 and 4. The inner member 16 includes the hub-wheelwheel hub 14 and the inner ring 10 press-fitted onto the hub-wheelwheel hub 14. The hub-wheelwheel hub 14 is integrally formed, at its outboard side, with a wheel mounting flange 6 on which, a wheel (not shown in Fig. 3) is mounted. An and with an inner raceway surface 14a is formed on the wheel hub 14 on of the outboard side of the bearing 15. Also, the wheel hub 14 and has the cylindrical portion 7 axially extending from the inner raceway surface 14a. The hub-wheelwheel hub 14 is formed with a serration (or spline) 8 on its inner circumferential surface (bore) to receive with a serration (or spline) 8 into which a serrated portion of the driving drive shaft (not shown in Fig. 3) is inserted so that at to transmit torque between the two can be transmitted therebetween.

[0044] The outer circumferential surface of the hub-wheelwheel hub 14 is formed with a flange portion 14b, corresponding to the large flange 10b of the inner ring 10, and a stepped portion 14c, to which abuts an inner end face (smaller end face) abuts. Thus, the so-called back-abutted type double row tapered roller bearing is formed structured. In addition, the The inner ring 10 is press-fitted onto the cylindrical portion 7 of the hub-wheelwheel hub 14. The inner ring 10 and is prevented from being axially slipped slipping off off from the cylindrical portion 7 by a caulked portion 13. The caulked portion 13 is formed by plastically deforming the end of the cylindrical portion 7 radially outward. Since this embodiment adopts at the self-retaining structure of such a third generation, it is not required to control an amount of preload in as a manner similar to the first embodiment by tightly fastening

a nut against the inner ring. Accordingly, it is possible to improve the readiness of assembly as well as to maintain the amount of preload for a long term.

**[0045]** Since the inner raceway surface 14a is directly formed on the outer circumferential surface of the ~~hub-wheel~~wheel hub 14, the rigidity of the ~~hub-wheel~~wheel hub 14 is increased. Accordingly, it is possible to reduce the size and weight and size of the bearing apparatus and to improve its even though although the ~~hub-wheel~~wheel hub 14 would be deformed by an a moment load applied applied thereto during running of the vehicle.

**[0046]** In this embodiment a A cap 17 is press-fitted into an opening of the ~~hub-wheel~~wheel hub 14 at its outboard side. This The cap 17 is made of austenitic-stainless steel sheet (JIS SUS 304 etc.) or preserved cold rolled steel sheet (JIS SPCC etc.) and includes~~comprises~~ a metal core 18. The cap 17 is formed as having~~to have~~ a substantially "C"-shaped cross-section. An and an elastic member 19, of rubber, is bonded via vulcanization, to the outer surface of the metal core 18. The metal core 18 is press-fitted~~fit~~ into the ~~hub-wheel~~wheel hub 14 so that the circumferential edge of the cylindrical fitting portion 18a is oriented toward the outboard side. This makes the press-fitting operation of the cap 17 easy. In addition, since the edge portion of the cap 17, having low rigidity, is positioned at the outboard side, the cap 17 does not move toward the outboard side. Thus, and thus it is possible to prevent the cap 17 from being slipped slipping off from the ~~hub-wheel~~wheel hub 14 even though~~although~~ the metal core 18 is deformed due to the elastic deformation of the ~~hub-wheel~~wheel hub 14.

**[0047]** As clearly shown in Fig. 4, a bead 18b ~~there~~ is formed at the circumferential edge. The ~~a~~ bead 18b ~~extending~~extends radially outward. It is possible to securely prevent axial movement of the cap 17 by engaging the bead 18b with an annular groove 20 formed on the inner circumferential surface (bore) of the ~~hub-wheel~~wheel hub 14.

**Third embodiment**

**[0048]** Fig. 5 is an enlarged partial view of a third embodiment of a vehicle wheel bearing apparatus for a ~~wheel of vehicle~~ of the present invention. The same ~~same~~ numerals are used herein as those used in the previous embodiments ~~for to~~ designate designating the same structural elements.

**[0049]** In this embodiment, a cap 21 is press-fitted ~~fit~~ into an opening of the ~~hub-wheel~~wheel hub 1 at its outboard side. This ~~The~~ cap 21 is made of austenitic-stainless steel sheet (JIS SUS 304 etc.) or preserved cold rolled steel sheet (JIS SPCC etc.) and includes~~comprises~~ a metal core 21a formed from ~~of~~ steel. The ~~cap 21 has formed as having~~ a substantially "C"-shaped cross-section. An, ~~and~~ an elastic member 21b extended from the inner circumferential surface to the cylindrical fitting portion of the metal core 21b. The elastic member 21b is formed from ~~comprises~~ a material such as rubber bonded, via vulcanization, to the surface of the metal core 21a. The elastic member 21b and ~~has~~ functions of preventing ~~to prevent~~ the generation of rust on the metal core 21a and sealing~~seals~~ the inside of the ~~hub-wheel~~wheel hub 1 from the outside with intimate contact of the elastic member 21b to the inner circumferential surface (bore) of the ~~hub-wheel~~wheel hub 1. Accordingly, it is possible to prevent ingress of rain water or dusts from ~~the~~ ambient

circumstances into the ~~driving~~drive shaft and thus into the differential gear oil. Also, it is possible to prevent ~~and~~ leakage of the differential gear oil to the outside.

**[0050]** The axial movement of the cap 21 is limited by a stop ring 22 secured on the inner circumferential surface of the ~~hub-wheel~~wheel hub 1, and a stepped portion 23. Thus, it is possible to prevent the cap 21 from slipping~~being slipped off from~~ ~~the hub-wheel~~wheel hub 1 even though~~although~~ the ~~hub-wheel~~wheel hub 1 is deformed by the repeated~~repeating~~ moment load applied~~applied thereto~~ during running of the vehicle. Projections 24, co-axially formed on the metal core 21a, increase the rigidity of the metal core 21a and improve the buckling resistance.

#### Fourth embodiment

**[0051]** Fig. 6 is an enlarged partial view of a fourth embodiment of the vehicle wheela bearing apparatus for a ~~wheel of vehicle~~ of the present invention. The same~~Same~~ numerals are used herein as those used in the previous embodiments for designating~~to designate~~ the same structural elements.

**[0052]** In this embodiment, an annular recess 25, having a circular arc cross section, is formed on the inner circumferential surface (bore) of the ~~hub-wheel~~wheel hub 1. A cap 26 is formed with a projection 26a having a cross section corresponding to the annular recess 25. The cap 26 is snapped ~~therein~~into the recess 25. This makes the mounting of the cap 26 easy and also~~enables~~ to prevent~~prevents~~ the axial movement of the cap 26 with a simple structure. In this case, it is unnecessary to form the projection 26a on the whole circumference of the cap 26. Thus, ~~and~~ three or more projections will sufficiently perform this function.

**[0053]** Fig. 7 is a partially enlarged longitudinal-section view of a modification of the fourth embodiment. ~~Same~~ The same numerals are used herein as those used in the previous embodiment (Fig. 6) for designating to designate the same structural elements.

**[0054]** A cap 27" has a cross section of substantially "C"-shaped configuration cross section and is press-fitted fit into the inner circumferential surface (bore) of the hub wheel wheel hub 1 with a predetermined interference. Then the The cap 27" is plastically deformed by a rolling tool and fitted into the annular recess 25. The formed projection 26a thus formed can further intimately contact the annular recess 25 of the hub wheel wheel hub 1 without rattle. Accordingly, it is possible to further effectively prevent the axial movement of the cap 27" and to securely prevent the leakage of the differential gear oil by this projection 26a in cooperation with the fitting portion 26b.

#### Fifth embodiment

**[0055]** Fig. 86 is an enlarged partial view of a fifth embodiment of the vehicle wheela bearing apparatus for a wheel of vehicle of the present invention. ~~The~~Same same numerals are used herein as those used in the previous embodiments for designating to designate the same structural elements.

**[0056]** In this embodiment, a cap 29 is press-fitted fit into the hub wheel wheel hub 1 over a region of the inner circumferential surface (bore) from its opened end at the outboard side to a pilot portion 28. The cap 29 is made of austenitic-stainless steel sheet (JIS SUS 304 etc.) or preserved cold rolled steel

sheet (JIS SPCC etc.). The cap 29 includes and comprises a metal core 29a press-formed ~~as having to have~~ a substantially "hat"-shaped cross-section. An, and an elastic member 29b, of rubber, is bonded, via vulcanization, over a region from the outer circumferential surface of the metal core 29a from the outer circumferential surface to the fitting portion and the outboard side end of the hub wheelwheel hub 1.

**[0057]** The elastic member 29b is formed of comprises rubber etc. and is being bonded to the metal core 29a, via e.g. vulcanization, and can perfectly prevent the leakage of differential gear oil and ingress of rain water or dusts into the differential gear oil through the drivingdrive shaft. In addition, since the cap 29 closes the whole the opened portion of the hub wheelwheel hub 1 and it is press-fitted fit into the portion of the hub wheelwheel hub 1 which is less deformed even although though the repeating repeated moment load is applied to the hub wheelwheel hub 1, it is possible to further prevent the elastic deformation of the cap 29 and its slipping off from the hub wheelwheel hub 1.

#### Applicability in industry

**[0058]** The vehicle wheel bearing apparatus for a wheel of vehicle of the present invention can be applied to a bearing apparatus for a wheel of vehicle of the driving wheel side of the semi-floating type wherein which a wheel bearing is arranged in opened portions between a drivingdrive shaft and a axle housing.

**[0059]** The present invention disclosure has been described with reference to the preferred embodiment. Obviously, modifications and alternations will occur to those of ordinary skill in the art upon reading and understanding the preceding

Attorney Docket No. 6340-000075/NP[INSERT HDP CLIENT MATTER NO.]

detailed description. It is intended that the present disclosure~~invention~~ be construed as including all such alternations and modifications insofar as they come within the scope of the appended claims or ~~the~~their equivalents~~thereof~~.

WHAT IS CLAIMED IS:

— 1. A vehicle wheel bearing apparatus for a wheel of vehicle comprising:  
— an axle housing “H” supported under a body of a vehicle;  
— a hollow drivingdrive shaft “D/S” inserted into the axle housing “H”;  
and  
— a wheel bearing arranged between the drivingdrive shaft “D/S” and an opening of the axle housing “H” and structured as a unit of a hub-wheelwheel hub (1, 14) and a double row rolling bearing (2, 15);  
— the wheel bearing comprising:  
— an inner member (3, 16) including a hub-wheelwheel hub (1, 14) integrally formed with a wheel mounting flange on one end thereof with a wheel mounting flange (6) and having an axially extending cylindrical portion (7);  
— and at least one inner ring (10) press-fitted onto the cylindrical portion (7) of the hub-wheelwheel hub (1, 14) and said at least one inner ring with at least one inner raceway surface formed on its outer circumferential surface with at least one of inner raceway surfaces (10a);  
— an outer member (4) arranged around the inner member (3, 16) and formed with double row outer raceway surfaces (4a) on its inner circumferential surface oppositely opposite to the inner raceway surfaces (10a);  
— double row rolling elements (5) arranged between the inner and outer raceway surfaces (10a, 4a) of the inner member (3, 16) and the outer member (4);  
— a cage (11) for freely rollably holding the rolling elements (5); and

—seals (12 and 13) for sealing an annular space between the inner member (3, 16) and the outer member (4); and

—characterized in that a cap (9, 17, 21, 26, 27', 29) having a metal core (9a, 18, 21a, 29a) of steel is press-fitted into an end of a central bore of the hub-wheelwheel hub (1, 14).

—2.—The vehicle wheelA bearing apparatus for a wheel of vehicle of claim 1 wherein said at least one (14a) of said inner raceway surfaces (10a) is formed directly on the outer circumferential surface of the hub-wheelwheel hub (1, 14).

—33.—The vehicle wheelA bearing apparatus for a wheel of vehicle of claim 1 wherein the end of said cylindrical portion (7) is plastically deformed radially outward to form a caulked portion (13) to prevent the inner ring (10) from being slipped off the cylindrical portion (7) of the hub-wheelwheel hub (1, 14).

—4.—A The vehicle wheel bearing apparatus for a wheel of vehicle of claim 3 wherein the outer circumferential region of the wheel mounting flange (6) from its base of an inboard side to the cylindrical portion (7) is hardened by high frequency induction hardening to have as having the a surface hardness of 54~64 HRC, and the caulked portion (13) is remained remains unhardened to have as having the surface hardness of 25 HRC or less after forging.

\_\_\_\_ —5. —The A-vehicle wheel bearing apparatus for a wheel of vehicle of claim 1 wherein said cap (9, 17, 21, 26, 27', 29) is press-fitted~~fit~~ into atthe central bore of the wheel mounting flange (6) of hub-wheelwheel hub.

(1, 14).

\_\_\_\_ —6. —The vehicle wheelA-bearing apparatus for a wheel of vehicle of claim 1 wherein said cap (9, 17, 21, 26, 27', 29) comprises a metal core (9a, 18, 21a, 29a) of steel having a ~~cross-section~~ of substantially "C" shaped configuration cross section and an elastic member (9b, 19, 21b, 29b) attached to at least part of its fitting portion..

\_\_\_\_ —7. —AThe vehicle wheel -bearing apparatus for a wheel of vehicle of claim 6 wherein said cap (9, 17, 21, 26, 27', 29) is press-fitted~~fit~~ so that the circumferential edge of its fitting portion is oriented toward the outboard side.

\_\_\_\_ —8. —The vehicle wheelA bearing apparatus for a wheel of vehicle of claim 6 wherein said circumferential edge of the fitting portion of the metal core (9a, 18, 21a, 29a) is formed with a radially outwardly extending bead~~bead~~ (18a) extending radially outward, and an annular groove (20) with which the bead (18a) engages is formed on the central bore of the hub-wheelwheel hub to engage the bead (1, 14).

\_\_\_\_ —9. —The vehicle wheelA-bearing apparatus for a wheel of vehicle of claim 6 wherein said cap (9, 17, 21, 26, 27', 29) is limited against an axial movement by steps (22, 23) provided at either sides of the cap (9, 17, 21, 26, 27', 29).

\_\_\_\_ —10.—The vehicle wheel A bearing~~bearing~~ apparatus for a wheel of vehicle of claim 1 wherein said cap {26} comprises a metal core of steel having a cross-section of substantially "C"-shaped configuration cross section, an annular recess {25} is formed on the inner circumferential surface of the hub ~~wheel~~wheel hub (1, 14), and the fitting portion of the cap {26} is formed with a projection {26a} adapted to be engaged with the annular recess {25}.

\_\_\_\_ —11.—The vehicle wheel A bearing apparatus for a wheel of vehicle of claim 10 wherein said projection {26a} is formed by plastic deformation after the cap {27} has been press-fitted~~fit~~ into the bore of the hub ~~wheel~~wheel hub (1, 14).

\_\_\_\_ —12.—The vehicle wheel A bearing apparatus for a wheel of vehicle of claim 1 wherein said cap {9, 17, 21, 26, 27', 29} is press-fitted~~fit~~ with an interference of 0.05~0.3 mm.

## ABSTRACT OF DISCLOSURE

An object of the present invention is to provide a A vehicle wheel bearing apparatus for a wheel of vehicle which can reduce reduces the weight, size and a number of parts and also can prevent prevents ingress of rain water or dusts and leakage of differential gear oil has. According to the present invention there is provided a bearing apparatus for a wheel of vehicle comprising: an axle housing supported under a body of a vehicle. A hollow drivingdrive shaft is inserted into the axle housing. A; and a wheel bearing is arranged between the drivingdrive shaft and an opening of the axle housing and is structured as a unit including of a hub wheelwheel hub and a double row rolling bearing. The; the wheel bearing includes comprising: an inner member with including a hub wheelwheel hub integrally formed with a wheel mounting flange on one end thereof with a wheel mounting flange and having an axially extending cylindrical portion. At least one; and inner rings- inner ring is press-fitted onto the cylindrical portion of the hub wheelwheel hub. The inner ring is and formed with at least one of the inner raceway surfaces on itswhich outer circumferential surface. An with at least one of inner raceway surfaces; an outer member is arranged around the inner member and formed with double row outer raceway surfaces on its inner circumferential surface oppositely opposite to the inner raceway surfaces;. Double double row rolling elements are arranged between the inner and outer raceway surfaces of the inner member and the outer member. A cage; a cage for freely rollably holding holds the rolling elements. Seals ; and seals seal for sealing an annular space between the inner member and the outer member. A cap, ; characterized in that a cap having a

Attorney Docket No. 6340-000075/NP~~INSERT HDP CLIENT MATTER NO.~~

metal core formed from~~of~~ steel, is press-fitted fit into an end of a central bore of the hub wheel~~wheel hub~~.